

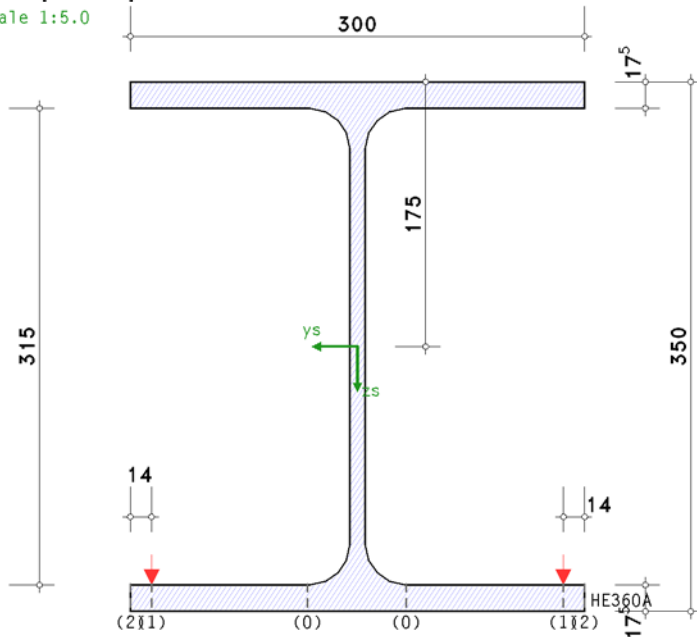
POS. 5: BSP. 3 WHEEL PRESSURE BOTTOM FLANGE

detailed problems acc. to Eurocode 3, EC 3-6 (12.10), NA: Deutschland

4H-EC3LK version: 11/2016-1f

1. input report

scale 1:5.0



steel grade

steel grade S235

cross-section

beam: section HE360A

loading

internal forces and moments at limit state of resistance (ULS):

Lk 1: EK 1

$$M_{y,Ed} = 174.7 \text{ kNm}, M_{z,Ed} = 6.2 \text{ kNm}$$

internal forces and moments at limit state of serviceability (SLS):

Lk 1: EK 12

$$M_{y,Ed} = 129.4 \text{ kNm}, M_{z,Ed} = 4.6 \text{ kNm}$$

transverse loading on lower edge of cross-section:

vertical wheel pressure $F_{z,Ed,ULS} = 9.23 \text{ kN}$, $F_{z,Ed,SLS} = 9.23 \text{ kN}$

distance of wheel axes $a_R = 100.0 \text{ cm}$

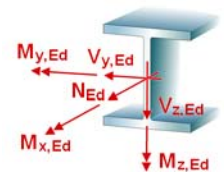
distance of wheel from lateral edge of flange $n_y = 14.0 \text{ mm}$

wheel at end of beam (unsupported lower flange, reinforced)

partial safety factors for material

resistance of cross-sections $\gamma_{M0} = 1.00$

serviceability $\gamma_{M,ser} = 1.00$



2. verification der local loading due to crane gantry

reinforcement of lower flange at end of beam: minimum dimensions $t_p = 17.5 \text{ mm} \times b_p = 300.0 \text{ mm}$

cross-sectional properties: $A = 142.76 \text{ cm}^2$, $z_s = 175.0 \text{ mm}$, $I_y = 33090.11 \text{ cm}^4$, $y_s = 0.0 \text{ mm}$, $I_z = 7886.85 \text{ cm}^4$

effective loading length from crane gantry:

$$l_{eff} = 2 \cdot (m+n) = 246.8 \text{ mm}$$

local stresses from crane gantry:

$$\sigma_{ux,Ed}(0) = 5.8 \text{ N/mm}^2, \sigma_{ux,Ed}(1) = 70.0 \text{ N/mm}^2, \sigma_{ux,Ed}(2) = 66.5 \text{ N/mm}^2$$

$$\sigma_{uy,Ed}(0) = -57.4 \text{ N/mm}^2, \sigma_{uy,Ed}(1) = 16.0 \text{ N/mm}^2, \sigma_{uy,Ed}(2) = 0.0 \text{ N/mm}^2$$

75% of local stresses from crane gantry:

$$\sigma_{ux,Ed}(0) = 4.3 \text{ N/mm}^2, \sigma_{ux,Ed}(1) = 52.5 \text{ N/mm}^2, \sigma_{ux,Ed}(2) = 49.9 \text{ N/mm}^2$$

$$\sigma_{uy,Ed}(0) = -43.1 \text{ N/mm}^2, \sigma_{uy,Ed}(1) = 12.0 \text{ N/mm}^2, \sigma_{uy,Ed}(2) = 0.0 \text{ N/mm}^2$$

2.1. resistance of lower flange (ULS)

permissible stress: $\sigma_{Rd} = f_y/\gamma_{M0} = 235.0 \text{ N/mm}^2$

Lk 1: $M_{y,Ed} = 174.7 \text{ kNm}$, $M_{z,Ed} = 6.2 \text{ kNm}$

normal stress $\sigma_{x,Ed} = 87.8 \text{ N/mm}^2$

$F_{z,Rd} = (I_{eff} \cdot t_{fu}^2 \cdot \sigma_{Rd}) / (4 \cdot m) \cdot [1 - (\sigma_{x,Ed} / \sigma_{Rd})^2] = 34.9 \text{ kN}$

$F_{z,Ed} = 9.2 \text{ kN} < F_{z,Rd} = 34.9 \text{ kN} \Rightarrow U = 0.264 < 1$ ok

maximum utilization: $\max U_{ULS} = 0.264 < 1$ ok

2.2. bending of lower flange (SLS)

permissible stress: $\sigma_{Rd} = f_y/\gamma_{M,ser} = 235.0 \text{ N/mm}^2$, $\tau_{Rd} = f_y/(3^{1/2} \cdot \gamma_{M,ser}) = 135.7 \text{ N/mm}^2$

Lk 1: $M_{y,Ed} = 129.4 \text{ kNm}$, $M_{z,Ed} = 4.6 \text{ kNm}$

(0) $\sigma_x = 73.1 \text{ N/mm}^2$, $\sigma_y = 43.1 \text{ N/mm}^2$, $\tau = 0.0 \text{ N/mm}^2$

normal stress $\sigma_x = 73.1 \text{ N/mm}^2 < \sigma_{Rd} = 235.0 \text{ N/mm}^2 \Rightarrow U = 0.311 < 1$ ok

normal stress $\sigma_y = 43.1 \text{ N/mm}^2 < \sigma_{Rd} = 235.0 \text{ N/mm}^2 \Rightarrow U = 0.183 < 1$ ok

equivalent stress $\sigma_{v1} = 73.1 \text{ N/mm}^2 < \sigma_{Rd} = 235.0 \text{ N/mm}^2 \Rightarrow U = 0.311 < 1$ ok

equivalent stress $\sigma_{v2} = 101.7 \text{ N/mm}^2 < \sigma_{Rd} = 235.0 \text{ N/mm}^2 \Rightarrow U = 0.433 < 1$ ok

utilization $U_{(0)} = 0.433 < 1$ ok

(1) $\sigma_x = 128.9 \text{ N/mm}^2$, $\sigma_y = 12.0 \text{ N/mm}^2$, $\tau = 0.0 \text{ N/mm}^2$

normal stress $\sigma_x = 128.9 \text{ N/mm}^2 < \sigma_{Rd} = 235.0 \text{ N/mm}^2 \Rightarrow U = 0.548 < 1$ ok

normal stress $\sigma_y = 12.0 \text{ N/mm}^2 < \sigma_{Rd} = 235.0 \text{ N/mm}^2 \Rightarrow U = 0.051 < 1$ ok

equivalent stress $\sigma_{v1} = 128.9 \text{ N/mm}^2 < \sigma_{Rd} = 235.0 \text{ N/mm}^2 \Rightarrow U = 0.548 < 1$ ok

equivalent stress $\sigma_{v2} = 123.3 \text{ N/mm}^2 < \sigma_{Rd} = 235.0 \text{ N/mm}^2 \Rightarrow U = 0.525 < 1$ ok

utilization $U_{(1)} = 0.548 < 1$ ok

(2) $\sigma_x = 127.1 \text{ N/mm}^2$, $\sigma_y = 0.0 \text{ N/mm}^2$, $\tau = 0.0 \text{ N/mm}^2$

normal stress $\sigma_x = 127.1 \text{ N/mm}^2 < \sigma_{Rd} = 235.0 \text{ N/mm}^2 \Rightarrow U = 0.541 < 1$ ok

equivalent stress $\sigma_{v1} = 127.1 \text{ N/mm}^2 < \sigma_{Rd} = 235.0 \text{ N/mm}^2 \Rightarrow U = 0.541 < 1$ ok

equivalent stress $\sigma_{v2} = 127.1 \text{ N/mm}^2 < \sigma_{Rd} = 235.0 \text{ N/mm}^2 \Rightarrow U = 0.541 < 1$ ok

utilization $U_{(2)} = 0.541 < 1$ ok

total: $U = 0.548 < 1$ ok

maximum utilization: $\max U_{SLS} = 0.548 < 1$ ok

3. final result

maximum utilization: $\max U = 0.548 < 1$ ok

verification succeeded