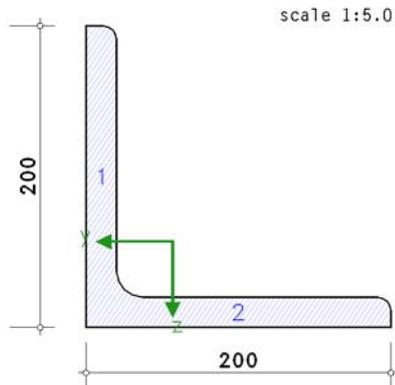


## POS. 18: KINDMANN/KRÜGER 2.8.6

verification of cross-section EC 3-1-8 (12.10), NA: Deutschland

### 1. input report



#### steel

steel grade S235

#### material safety factor

resistance of cross-sections  $\gamma_{M0} = 1.00$

#### geometry

section L 200 X 20

#### resistance

elastic verification, calculation as thin-walled cross-section

plastic verification with partial internal forces and moments

valid normal, shear-, equivalent stress:  $\sigma_{x,Rd} = 235.0 \text{ N/mm}^2$ ,  $\tau_{Rd} = 135.7 \text{ N/mm}^2$ ,  $\sigma_{v,Rd} = 235.0 \text{ N/mm}^2$

#### internal forces and moments referring to local axes of cross-section

$\sigma$ -generating forces (N, M) work at centroid,  $\tau$ -generating forces (V,  $T_t$ ) work at shear center

Lk 1:  $M_{y,Ed} = 4375.00 \text{ kNcm}$ ,  $V_{z,Ed} = 25.00 \text{ kN}$

#### notes

buckling is not investigated.

### 2. Lk 1

#### 2.1. verification of cross-section

internal forces and moments referring to local axes of cross-section:  $M_y = 4375.00 \text{ kNcm}$ ,  $V_z = 25.00 \text{ kN}$

internal forces and moments referring to main axes:  $M_{\eta} = 3093.59 \text{ kNcm}$ ,  $V_{\zeta} = 17.68 \text{ kN}$ ,  $M_{\zeta} = -3093.59 \text{ kNcm}$ ,  $V_{\eta} = 17.68 \text{ kN}$

Hauptträgheitsmomente  $I_{\eta} = 4529.01 \text{ cm}^4$ ,  $I_{\zeta} = 1172.02 \text{ cm}^4$ ,  $\alpha = 45.00^\circ$

##### 2.1.1. elastic verification

elastic verification for  $M_{\eta} = 43.75 \text{ kNm}$ ,  $V_{\zeta} = 25.00 \text{ kN}$

elastic stresses:  $\max |\sigma_x| = 273.37 \text{ N/mm}^2$ ,  $\max \tau = 8.88 \text{ N/mm}^2$ ,  $\max \sigma_v = 273.37 \text{ N/mm}^2$

$\max \sigma_x$  bei  $y = 56.8 \text{ mm}$ ,  $z = 56.8 \text{ mm}$ :  $\sigma_x = 212.39 \text{ N/mm}^2$ ,  $\tau = 4.93 \text{ N/mm}^2$ ,  $\sigma_v = 212.57 \text{ N/mm}^2$

$\min \sigma_x$  bei  $y = 45.8 \text{ mm}$ ,  $z = -143.2 \text{ mm}$ :  $\sigma_x = -273.37 \text{ N/mm}^2$ ,  $\tau = 0.00 \text{ N/mm}^2$ ,  $\sigma_v = 273.37 \text{ N/mm}^2$

$\max \tau$  bei  $y = 46.8 \text{ mm}$ ,  $z = -29.2 \text{ mm}$ :  $\sigma_x = -3.79 \text{ N/mm}^2$ ,  $\tau = 8.88 \text{ N/mm}^2$ ,  $\sigma_v = 15.84 \text{ N/mm}^2$

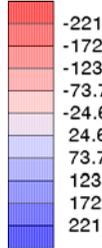
$\max \sigma_v$  bei  $y = 45.8 \text{ mm}$ ,  $z = -143.2 \text{ mm}$ :  $\sigma_x = -273.37 \text{ N/mm}^2$ ,  $\tau = 0.00 \text{ N/mm}^2$ ,  $\sigma_v = 273.37 \text{ N/mm}^2$

valid equivalent stress:  $\sigma_{v,Rd} = 235.0 \text{ N/mm}^2$

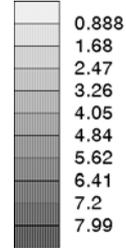
verification:  $\sigma_v = 273.37 \text{ N/mm}^2 > \sigma_{v,Rd} = 235.00 \text{ N/mm}^2 \Rightarrow U_{\sigma} = 1.163 > 1$  **fault !!**

c/t-ratio: outstand flange: utilization  $U_{c/t} = 0.546 < 1$  **ok**

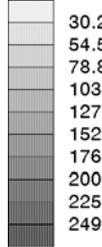
normal stresses  $\sigma_x$  [N/mm<sup>2</sup>]  
 min  $\sigma_x = -276.3$ , max  $\sigma_x = 212.1$



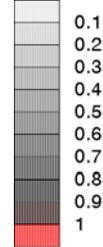
shear stresses  $\tau$  [N/mm<sup>2</sup>]  
 max  $\tau = 8.9$



equivalent stresses  $\sigma_v$  [N/mm<sup>2</sup>]  
 max  $\sigma_v = 276.3$



utilization  $U_\sigma$   
 max  $U_\sigma = 1.176$



stresses, utilizations

y mm	z mm	$\sigma_x$ N/mm <sup>2</sup>	$\tau$ N/mm <sup>2</sup>	$\sigma_v$ N/mm <sup>2</sup>	$U_\sigma$
-15.1	1.0	-276.28	0.15	276.28	1.176 > 1
0.0	200.0	212.14	3.83	212.25	0.903
-5.0	115.2	5.99	8.88	16.51	0.070

y,z: node coordinates;  $\sigma_x, \tau, \sigma_v$ : stresses;  $U_\sigma$ : stress utilization

### 2.1.2. plastic verification

plastic verification for  $M_\eta = 43.75$  kNm,  $V_\zeta = 25.00$  kN

valid normal/shear stress:  $\sigma_{x,Rd} = 235.0$  N/mm<sup>2</sup>,  $\tau_{Rd} = 135.7$  N/mm<sup>2</sup>

bottom flange: resistance forces  $N_{max,U} = 418.64$  kN,  $N_{min,U} = -418.64$  kN

web: shear force  $V_s = 25.00$  kN, shear stress  $\tau_s = 6.94$  N/mm<sup>2</sup>  $\Rightarrow U_{\tau,s} = 0.051$

resistance forces  $N_{max,S} = 844.89$  kN,  $N_{min,S} = -844.89$  kN

main bending: moment  $M_y = 43.75$  kNm, resistance moments  $M_{y,max} = 70.55$  kNm,  $M_{y,min} = -70.55$  kNm  $\Rightarrow U_{M_y} = 0.620$

total (possibly due to load increase): max  $U = 0.623 < 1$  **ok**

c/t-ratio: outstand flange: utilization  $U_{c/t} = 0.578 < 1$  **ok**

### 3. final result

maximum utilization:

stress	max $U_\sigma = 1.163 > 1$ <b>fault !!</b>
c/t-ratio	max $U_{c/t} = 0.578 < 1$ <b>ok</b>
resistance	max $U = 1.163 > 1$ <b>fault !!</b>

**resistance not ensured !!**